

11. ~~602~~ between a first braking state corresponding to a first amplification ratio of a braking force and a second braking state corresponding to a second amplification ratio of the braking force, and a deformable reaction disc interposed between the piston and the plunger and said reaction rod,

wherein said thrust assembly includes means for absorbing deformations of the reaction disc, said deformations contributing to determining a braking force at which passage from the first braking state to the second braking state takes place.

16. Amplifier as described in claim 1, wherein the deformations are a function of stiffness of the reaction disc.

17. Amplifier as described in claim 1, wherein the piston includes a piston bearing surface in physical contact with a thrust assembly bearing surface, wherein an area of contact between the piston bearing surface and the thrust assembly bearing surface determines the second amplification ratio, and a distance (X) between the piston bearing surface and the thrust assembly bearing surface, at rest, determines the braking force at which the passage from the first braking state to the second braking state takes place.

18. Amplifier as described in claim 1, wherein the piston bearing surface and the thrust assembly bearing surface are annular and co-axial.

19. Amplifier as described in claim 1, wherein the reaction disc is arranged in a housing formed in a head at an end of the reaction rod, said housing providing a cavity forming said means for absorption of said deformations.

20. Amplifier as described in claim 5, wherein the housing is suitable to receive an end of the piston, said housing having a dimension greater than that of the end of the piston to provide the cavity for absorption of the deformations of the reaction disc.

21. Amplifier as described in claim 4, wherein the reaction rod includes at an end a head mounted to be axially displaceable within a sleeve provided with a flange forming an orifice in which an end of the piston slides, and a bearing surface the reaction disc being housed within the sleeve, said reaction disc bearing on the head and on the bearing surface of the flange, said bearing surface of the flange co-operating with the end of the piston on displacement of the piston to form said means for absorption of the deformation of the reaction disc.

22. Amplifier as described in claim 7, wherein the orifice formed by the flange of the sleeve is circular and the end of the piston able to slide in said orifice is cylindrical.

23. Amplifier as described in claim 8, wherein elastic means are mounted within the sleeve bearing on a face of the head of the reaction rod opposite to the reaction disc and on a stop operably attached to the sleeve to apply a return force to said sleeve.

24. Amplifier as described in claim 9, wherein the elastic means comprise a conical washer.

25. Amplifier as described in claim 10, wherein the sleeve includes a shoulder operable to be a travel limit stop to the head of the reaction rod in the absence of braking.

26. Amplifier as described in claim 11, wherein the reaction disc is incompressible.

27. Amplifier as described in claim 12, wherein the first amplification ratio is constant.

28. Amplifier as described in claim 13, wherein the second amplification ratio is constant.

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